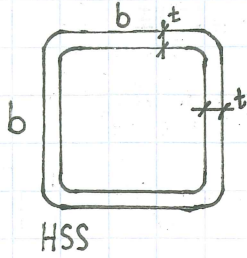


CIV02 - STRUCTURES and MATERIALS

Topic: Thin Plate Buckling

1) Thin Wall Members

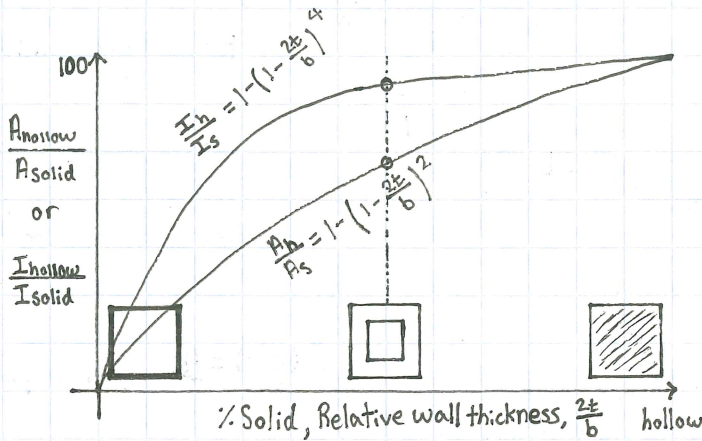


$$\text{Area} = b^2 - (b-2t)^2 \quad I = \frac{b^4}{12} - \frac{(b-2t)^4}{12}$$

$$\frac{A_{\text{hollow}}}{A_{\text{solid}}} = \frac{b^2 - (b-2t)^2}{b^2} = 1 - \left(1 - \frac{2t}{b}\right)^2 \quad (\text{weight related to this})$$

$$\frac{I_{\text{hollow}}}{I_{\text{solid}}} = \frac{\frac{b^4}{12} - \frac{(b-2t)^4}{12}}{\frac{b^4}{12}} = 1 - \left(1 - \frac{2t}{b}\right)^4 \quad (\text{Strength related to this})$$

Strength = quartic



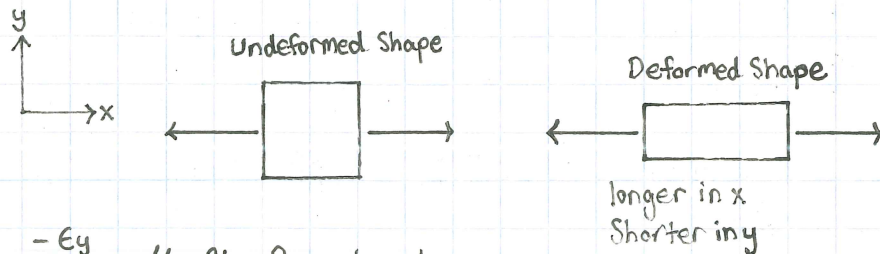
$$t = \frac{b}{2} \rightarrow \text{Empty}$$

Thinner walled members have better strength to weight ratios

2) Plate Buckling

- Will limit how thin the wall can be
- Derivation is hard

2a) Poisson's Ratio

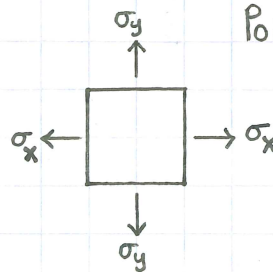


$$\frac{-\epsilon_y}{\epsilon_x} = \mu = \nu \quad \text{Poisson's ratio is constant}$$

rubber, $\mu = 0.5$

Steel, $\mu = 0.3$

Cork, $\mu = 0$



Strain from Stress adds to strain from Poisson's Ratio

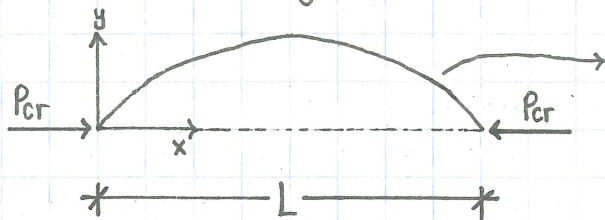
$$\epsilon_x = \frac{\sigma_x}{E} - \mu \frac{\sigma_y}{E} \quad \epsilon_y = \frac{-\mu \sigma_x}{E} + \frac{\sigma_y}{E}$$

$$\sigma_x = \frac{E}{1-\mu^2} (\epsilon_x + \mu \epsilon_y)$$

$$\sigma_y = \frac{E}{1-\mu^2} (\mu \epsilon_x + \epsilon_y)$$

Effective E?

2b) Euler Buckling



$$y(x) = A \sin\left(\frac{n\pi x}{L}\right)$$

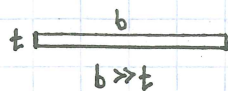
integer

$$P_{cr} = \frac{\pi^2 EI}{L^2}$$

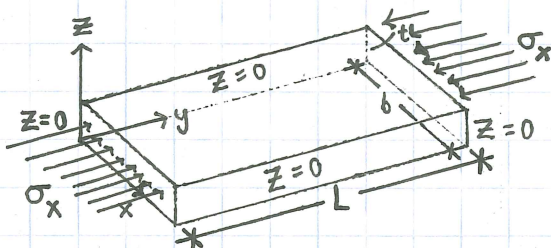
$$\frac{P_{cr}}{A} = \frac{\pi^2 E b t^3}{12 L^2 b t}$$

$$\sigma_{crit} = \frac{\pi^2 E}{12} \left(\frac{t}{L}\right)^2 \quad \text{Global Buckling/Euler Critical Stress}$$

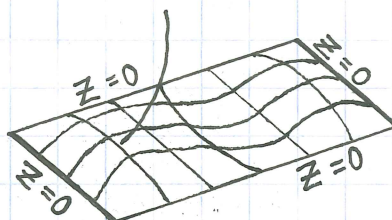
Cross section



2c) Local Buckling Equation



Buckles Between Boundaries

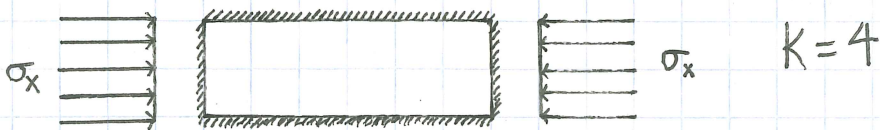


$$z(x,y) = A \sin\left(\frac{m x \pi}{L}\right) \sin\left(\frac{n y \pi}{L}\right) \quad m, n \text{ are } 1, 2, 3, \dots$$

3) Thin Plate Buckling Equation

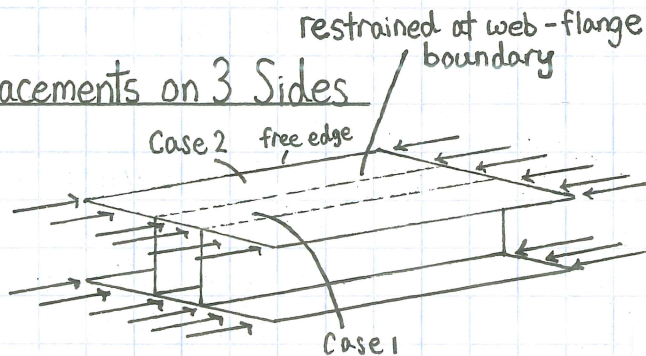
$$\sigma_{cr} = \frac{K \pi^2 E}{12(1-\nu^2)} \left(\frac{t}{b}\right)^2 \quad \text{if } L < b \text{ use } \left(\frac{t}{L}\right)^2$$

Case 1 - Restrained Against z Displacements on all 4 Sides

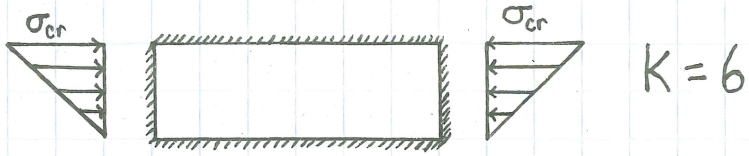


Case 2 - Restrained Against z Displacements on 3 Sides

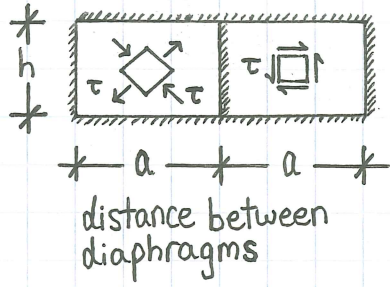
$$K = 0.425$$



Case 3



Case 4 - Shear Buckling



$$\tau_{cr} = \frac{5\pi^2 E}{12(1-\mu^2)} \left[\left(\frac{t}{h}\right)^2 + \left(\frac{t}{a}\right)^2 \right]$$