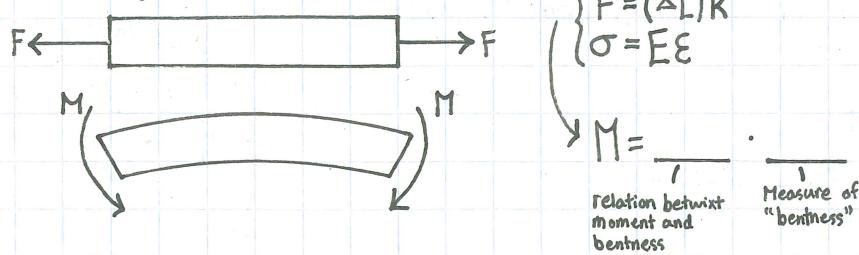


# CIVI02 - STRUCTURES and MATERIALS

Topic: Bending of Beams

## 1) Bending



$$\begin{cases} F = (\Delta L)K \\ \sigma = E\epsilon \end{cases}$$

$$M = \frac{\sigma}{E} \cdot \frac{I}{I}$$

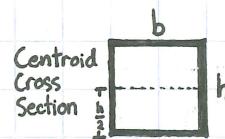
relation between moment and bentness  
Measure of "bentness"

Assume

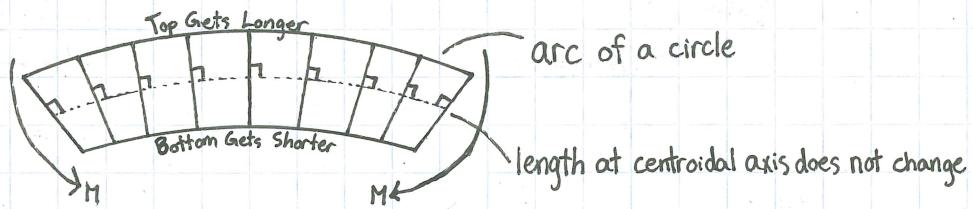
Hooke's Law

"Plane Sections Remain Plane" → Euler (Swiss), Bernoulli, Navier (French), Timochenko, Hooke's (English, 1678)

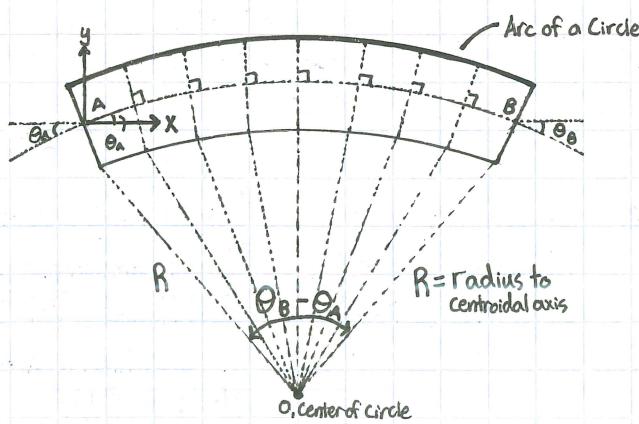
Undeformed Shape



Deformed Shape



## 2) Flexural Stiffness



Note: Diagram Not to Scale. Lines should be straight.

$L_{AB}$  = Length Along Arc

$$L_{AB} = (\theta_B - \theta_A) \cdot R$$

From Hooke, we will assume this is constant.

If  $\theta_A$  and  $\theta_B$  are small, then the horizontal component of length  $L_{AB}$  will equal  $L_{AB}$ .

Define degree of "bentness" as curvature.

$$\phi = \frac{(\theta_B - \theta_A)}{L_{AB}}$$

= Rate of Change of Angle = Slope of Beam

$$\phi = \frac{d\theta}{dx}$$

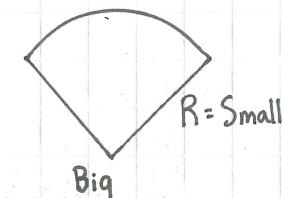
$$\phi = \frac{(\theta_B - \theta_A)}{L_{AB}} = \frac{(\theta_B - \theta_A)}{\underbrace{L_{AB}}_{\text{Whole Thing}}} = \frac{(\theta_B - \theta_A)}{\underbrace{L_{AB}}_{\text{Small Parts}}}$$

$$L_{AB} = (\theta_B - \theta_A) \cdot R$$

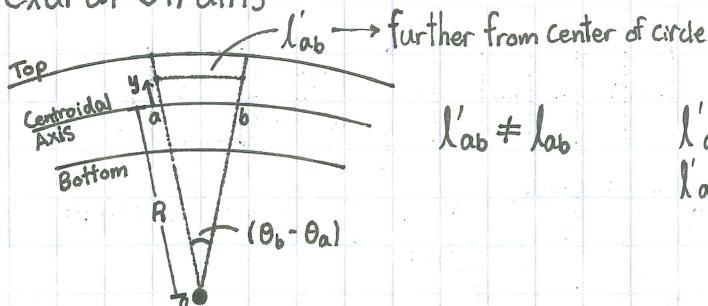
$$R = \frac{L_{AB}}{(\theta_B - \theta_A)}$$

$$\phi = \frac{(\theta_B - \theta_A)}{L_{AB}}$$

$$\phi = \frac{1}{R}$$



### 3) Flexural Strains



$$l'_AB \neq l_{AB}$$

$$\epsilon = \frac{\Delta L}{L}, \Delta L = l'_{AB} - l_{AB}$$

$$\epsilon = \frac{\phi l_{AB} \cdot y}{l_{AB}}$$

$$\boxed{\epsilon = \phi \cdot y}$$

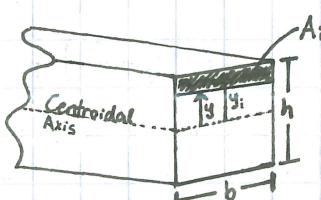
$$\begin{aligned} l'_{AB} &= l_{AB} \\ l'_{AB} &= (\theta_B - \theta_A) \cdot (R + y) \\ &= (\theta_B - \theta_A) \cdot \left(\frac{l}{\phi} + y\right) \end{aligned}$$

$$\phi = \frac{(\theta_B - \theta_A)}{l_{AB}} \rightarrow (\theta_B - \theta_A) = \phi l_{AB}$$

$$l'_{AB} = \phi l_{AB} \left(\frac{1}{\phi} + y\right)$$

$$l'_{AB} = l_{AB} + \phi l_{AB} y$$

### 4) Relation of M to φ



$$\begin{aligned} \text{Strain} &= \epsilon = \phi y_i \\ \text{Stress} &= \sigma_i = E \phi y_i \\ \text{Force} &= F_i = E \phi y_i A_i \end{aligned}$$

Moment About Centroid = Force · Lever Arm

Moment Component =  $E \phi y_i A_i y_i$

Total Moment =  $\sum_i E \phi y_i^2 A_i$

take limit as  $A_i \rightarrow 0$

$$\begin{aligned} M &= \int E \phi \cdot y^2 dA \\ &= E \phi \underbrace{\int y^2 dA}_{\text{Second Moment of Area} = I} \end{aligned}$$

$\boxed{M = EI\phi}$

$$\boxed{M = EI\phi}$$